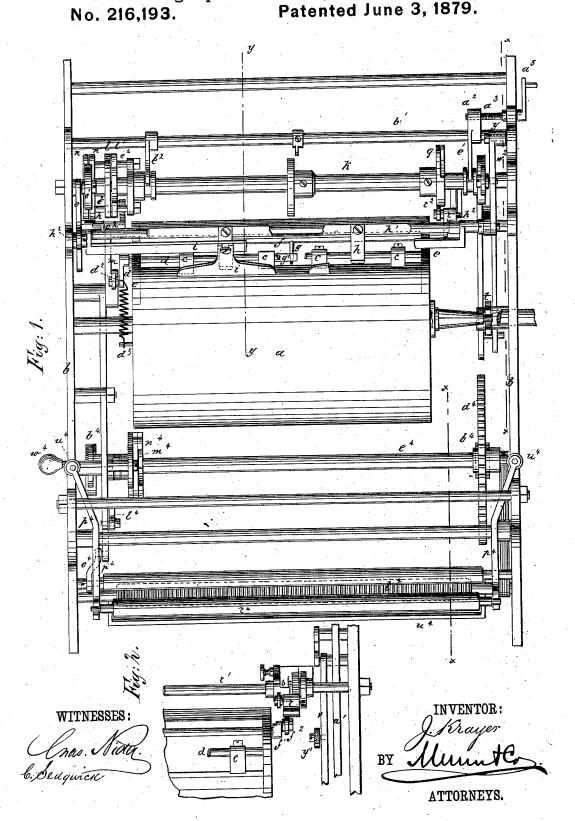
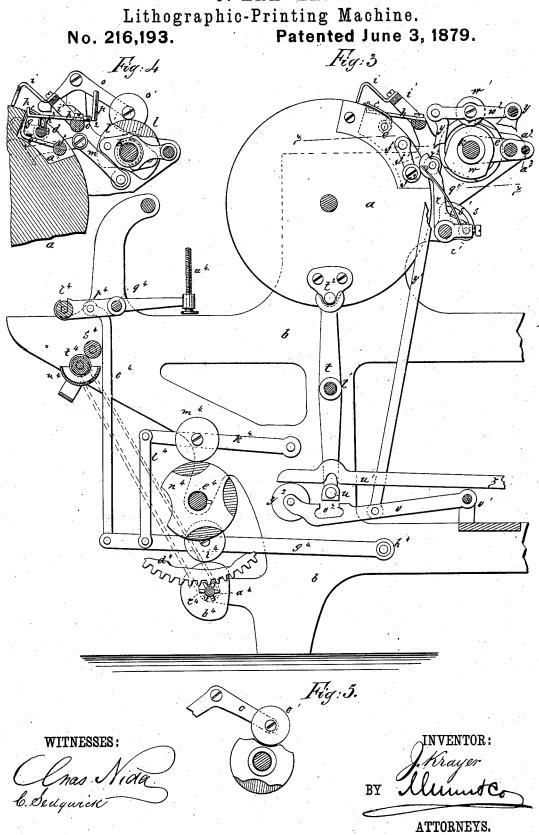
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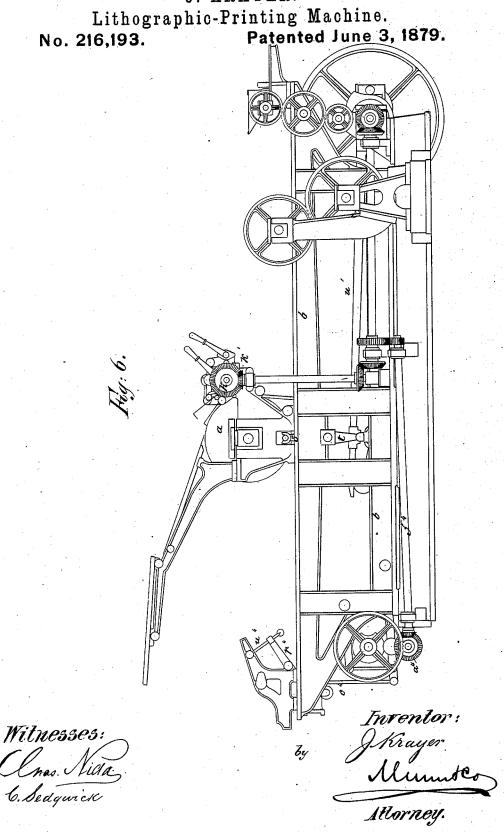
Lithographic-Printing Machine.
No. 216,193. Patented June 3, 1879.



## J. KRAYER.



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## UNITED STATES PATENT OFFICE.

JOSEPH KRAYER, OF JOHANNISBERG-ON-THE-RHINE, WIESBADEN, GERMANY.

## IMPROVEMENT IN LITHOGRAPHIC PRINTING-MACHINES.

Specification forming part of Letters Patent No. **216,193**, dated June 3, 1879; application filed October 10, 1878.

To all whom it may concern:

Be it known that I, JOSEPH KRAYER, of Johannisberg-on-the-Rhine, Wiesbaden, Germany, have invented new and useful Improvements in Lithographic Printing-Machines, of which the following is a specification.

The invention will first be described in connection with the drawings, and then pointed

out in the claims.

In the accompanying drawings, Figure 1 is a plan view of the cylinder and frame of a lithographic press having my improvements applied thereto. Fig. 2 is a section plan at one end of the cylinder on the line zz. Fig. 3 is a side elevation with the side frame removed on line xx. Fig. 4 is a cross-section of the cylinder and paper-regulating devices, taken on line yy. Fig. 5 is a detail view, separately referred to; and Fig. 6 is a side elevation of a press, showing the connection of my improvements with the other parts of the press.

Similar letters of reference indicate corre-

sponding parts.

The cylinder a and frame b are of usual construction. The press to which my improvements are applied is of the class in which the stone is carried by a reciprocating carriage. The press and its operating mechanism are represented in Fig. 6, and the application of my improvements to the same will be understood from the following description.

c c c c are the gripers carried by a rock-shaft, d, that is within a cavity,  $a^1$ , cut across the face of cylinder a, and journaled in the plates e at the ends of the cavity. One end of shaft d extends outside plate e, and is provided with a crank-arm,  $d^1$ , and friction-roller,  $d^2$ . (See Fig. 1.) The gripers e are kept in contact with cylinder a by a spring,  $d^3$ , attached to arm  $d^1$  and cylinder a.

f is a second rock-shaft within the cavity  $a^1$ , carrying the point-spur g, (see Figs. 1 and 4,) and it has at one end the crank-arm  $f^1$  and

friction-roller  $f^2$ . (See Fig. 3.)

h h are the guides against which the edge of the sheet is to be placed. i i are the planishers, which rest upon the paper that is on the cylinder a, to prevent the formation of creases and folds. The guides h are upon a rod,  $h^1$ , that is held loosely in the sockets  $h^2$ , held in frame b, so that these guides fall by their own

weight below the surface of cylinder a, their downward motion being limited by a pin,  $h^3$ , that rests on shaft d. The planishers i are attached to a bar,  $i^1$ , that is also fitted loosely in the sockets  $h^2$ , so that the planishers rest by their own weight on the paper or cylinder.

k is a shaft mounted in an extension of frame b, in front of cylinder a. l  $l^1$  are camdisks upon the shaft k, for operating the gripers c through the medium of the elbow-lever m. Cam l has but one noteh or depression, while  $l^1$  has two notches, so that by one cam the gripers will be operated once during a revolution of shaft k, and by the other cam they will be lifted twice. The manner of shifting the cams will be hereinafter described.

The angle-lever m is fulcrumed on frame a. Its short arm carries a friction-roller that rests against the under side of cams l l, and its long arm is brought by the cams into contact with the roller d on arm d of griper-shaft d.

n  $n^1$  are cam-disks on shaft k, which act, through the medium of elbow-lever o, to lift the planishers i and then permit them to fall. The cam n acts once during the revolution of shaft k, while cam  $n^1$  acts twice. The lever o carries a friction-roller,  $o^1$ , which rests on cam n or  $n^1$ , (see Fig. 5,) and its short arm acts upon a stud on bar  $i^1$  to turn the same in the sockets  $h^2$  and raise the planishers.

p is a pin projecting downward from the lever o, and coming in contact with a projection,  $o^2$ , from the rod  $h^1$ , that carries the guides h, so that when the roller  $o^2$  falls into one of the notches of cams n  $n^1$  the guides h h are thrown

up.

q is a cam-disk on shaft k for operating the point-spur g through the medium of a lever, r, that is hung on the cross-bar  $r^1$ . The lever r carries a friction-roller,  $r^2$ , that is kept in contact with cam q by a spring,  $q^1$ . The cam q is double, one portion of its surface having diametrically-opposed depressions, and the other portion but one depressing, so that by shifting cam q on shaft k the roller  $r^2$ , and through it the point g, are given a multiple action during a revolution of shaft k, or the lever r may be entirely freed from contact with cam q.

The lever r acts upon the roller  $f^2$  of crank  $f^1$  to raise spur g above the surface of cylinder a and cause it to pierce the sheet, and the

spur is pressed down again by a spring,  $g^1$ , as I soon as the roller  $r^2$  falls into the depressions

The spur g is also caused to pierce a hole in the sheet after it has been seized by the gripers c and carried a short distance by means of the block s, that is fitted to swing upon the bar  $r^1$ , and thereby be moved in or out of the path of roller  $f^2$ . This block s is held in either position by a clamping-screw, which passes through a slot,  $s^1$ , in block s.

The shaft k is to be revolved continuously by bevel-gearing  $k^1$ , as shown in Fig. 6. The stoppage of cylinder a, when but one impression is to be taken during a movement to and fro of the carriage, is effected by the devices

shown in Fig. 3.

t is a forked catch-lever hung at  $t^{1}$  on frame b, the upper end of which engages with a pin,  $t^2$ , on cylinder a. Its lower end carries a crankpin, u, which engages with the connecting-rod u'. v' is an arm hung at  $v^1$  and formed with a mortise,  $v^2$ , beneath the lower end of lever t, which mortise is connected with or relieved from lever t by the cam w on shaft k, acting upon a roller,  $w^1$ , on the lever  $w^2$ , that is pivoted at y, and is connected by a rod,  $y^1$ , to the arm v. The cam w raises arm v, thereby disconnecting the rod u' from crank-pin u by means of roller  $y^2$ , and locking the catch-lever t by means of mortise  $v^2$  once during each revolution of shaft k.

The cams l, n, and w are shifted simultaneously to render the action of the devices described single or double during the revolution of shaft k by means of a screw,  $a^3$ , fitted in frame b and working in a projection,  $a^2$ , from a rod,  $b^1$ , to move the rod  $b^1$  endwise in its bearings in frame b, which endwise motion, by means of forks  $e^1$ , that enter grooves formed in the hubs of cams l and w, moves the said cams upon shaft k. The cam l is connected by rods  $e^2$  with cam n, so that they move together, and the screw  $a^3$  is provided with a handle,  $a^5$ , by which it is operated.

When the cam l is in contact with  $l^1$ , and cam n with  $n^1$ , the guides h, planishers i, and gripers c will operate but once during the revolution of shaft k, the single-acting cams being then the operative ones. This single action is used when but one impression is to be taken during a double movement of the carriage. The  $\overline{\operatorname{cam}} w$  will be in position at this

time to stop the cylinder a, as before described. The separation of the cams by the screw  $a^3$ , as before described, permits the double-acting cams  $l^1$   $n^1$  to operate, and moves cam w out of

action.

The moistening devices will be next described. The shaft  $a^4$ , journaled in frame b, carries a cam,  $b^4$ , and gear-pinion  $c^4$ , which latter meshes with the gear-wheel d4 on shaft e4. Shaft a4 is driven by gearing on a shaft,  $f^4$ , Fig. 6, which takes its motion from the other mechanism of the press, and the connection is to be such that the shaft a4 will revolve once during each reciprocation of the stone,

and once to every eight revolutions of the shaft  $e^4$ .  $g^4$  is a lever hung at  $h^4$ , and carrying a roller,  $i^4$ , that rests on cam  $b^4$ .  $k^4$  is a second lever connected to  $g^4$  by a link,  $l^4$ , and carrying a roller,  $m^4$ , that rests on a portion of the three-part cam  $n^4$  on shaft  $e^4$ .  $o^4$  is a rod connecting lever  $g^4$  with one of the levers  $p^4$ , that are on a rock-shaft,  $q^4$ , and earry the roll  $r^4$ .  $s^4$  is a roller that is in contact with a roller,  $t^4$ , which latter turns in a vessel,  $u^4$ , for containing water. The rollers  $r^4$  and  $s^4$  should be covered with cloth or felt to carry the water from roller  $t^4$  to roller  $s^4$  and by the roller  $r^4$  to the moistening-table and rolls. (Not shown.)

The amount of pressure of roller  $r^4$  upon roller  $s^4$  is regulated by the screws  $u^4$ , which are on the outer ends of levers  $p^4$  and take against the frame b. The cam  $n^4$  is capable of sidewise motion on its shaft, and is caused to turn therewith by a feather on the shaft. The cam  $n^4$  can be moved on its shaft by a pin and fork,  $w^4$ , so as to bring the roller  $m^4$  upon any one of the three disks of the cam or relieve it entirely from contact with  $n^4$ , in which latter case the motion of roller  $r^4$  will be given solely by cam  $b^4$  on shaft  $a^4$ . This cam  $b^4$  is shaped so that it operates the roll  $r^4$  once during the revolution of  $a^4$ , and consequently the roll  $r^4$ will be moved once during each reciprocation of the stone. On the other hand, if the cam  $n^4$  be positioned so that the roller  $m^4$  rests upon the disk of cam  $n^4$ , containing four notches, the roll  $r^4$  will be operated every second reciprocation, and if upon the disk containing two notches, every fourth reciprocation. The pin  $w^4$  is to be elamped to retain the cam  $n^4$  in place by a thumb-screw.

Having thus described my invention, I claim as new and desire to secure by Letters Pat-

ent-

1. The combination, with shaft d, having gripers and arm  $d^{i}$ , of the lever m, the laterally adjustable cam l, having one notch, and the fixed cam l1, having two notches, as shown and described, for the purpose set forth.

2. The mechanism for operating the planishers i and guides h, consisting of the cams n  $n^{i}$ , lever o, bar i, pin p, projection  $o^{2}$ , and rod  $h^1$ , combined and arranged substantially

as described.

3. The combination, with the gripers, guides, planishers, and lever v, of the shaft k and fixed cams  $l^1$  and  $n^1$ , the movable cams l, n, and w, rod  $b^1$ , forks  $e^1$ , and screw  $a^3$ , arranged for operation substantially as described, and for the purposes set forth.

4. In combination with the shaft  $a^4$ , cam  $b^4$ , and arm  $g^4$ , by which the moistening-roll  $r^4$  is operated at each revolution of shaft  $a^4$ , the three-part cam  $n^4$ , arm  $k^4$ , link  $l^4$ , and pin and fork  $w^4$ , substantially as and for the purposes

described.

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Witnesses:

Aug. Diesterweg, Jos. KLEIN.