J. TRIPP.

MECHANICAL MOVEMENT.

No. 304,048.

Patented Aug. 26, 1884.

Fig.1,

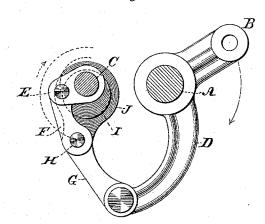


Fig. 2,

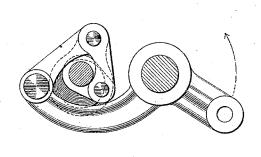


Fig. 4,

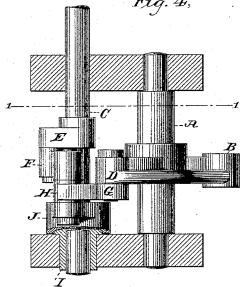
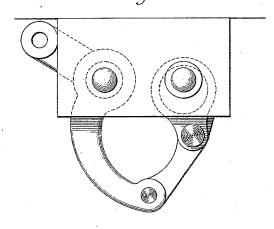


Fig. 3,



Witnesses:

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UNITED STATES PATENT

JAMES TRIPP, OF NEW YORK, N. Y.

MECHANICAL MOVEMENT.

CPECIFICATION forming part of Letters Patent No. 304,048, dated August 26, 1884.

Application filed June 24, 1884. (No model.)

To all whom it may concern:

Be it known that I, JAMES TRIPP, of the city, county, and State of New York, have invented certain new and useful Improvements in Me-5 chanical Movements; and I do hereby declare the same by the following full, clear, and exact description thereof, reference being had to

the accompanying drawings.

This motion is applicable to any machine 10 where an angular, reciprocating, or oscillating motion is required. It is, however, particularly adapted to oscillate the circularly-moving shuttles of sewing-machines, and I illustrate it in the accompanying drawings as con-15 structed and proportioned for that parpose. With such shuttles it is requisite that they shall oscillate through more than one hundred and eighty degrees, and it is desirable, in order to best accomplish their work, that they shall 20 move through even a considerably larger angle of oscillation than one hundred and eighty degrees—that is, at least two hundred and fifty-five degress, or a maximum of about two hundred and eighty. This invention, there-25 fore, consists of an improved mechanical movement, whereby a circular or angular oscillatory motion extending through threequarters of a circle, and more, can be given to a shaft.

Referring to the drawings, Figures 1 and 2 are side views of mechanism embodying the invention, and respectively illustrate the parts in the positions that they occupy near the beginning and near the end of the range of move-35 ment; and Fig. 3 is a similar view from the opposite side, Fig. 4 being a plan view of the same, looking upward, and the plane 17 indieating the sections of Figs. 1 and 2.

In these drawings, A represents what I shall 40 term the "main" or "driving" rock-shaft, to which the power is directly transmitted, as through the medium of arm B, or otherwise; and C is the driven rock-shaft that is to be oscillated by the main shaft but through a much 45 greater arc than that of the main shaft.

D is a vibrating arm, fast to the driving-shaft. E is a crank-arm fast to the driven rock-shaft, and F and G are links, one end of each of which are jointed together at H, and the other 50 ends of which are jointed, one to the arm D, and the other to the crank-arm E.

rock-shaft C, as seen in Fig. 3. In the present case this block is preferably arranged to constitute one of the bearings of the rock-shaft 55 C, so as to give rigidity to the parts; but this is not essential to the construction, as this rock-shaft may end with its crank-arm. This eccentric block carries the guide arm J, which at its outer end is jointed to the links by the 60 pivot-pin H, that connects them. The relative positions of the eccentric block and driven rock-shaft C, or of the pivotal center of the guide-arm and the center of this rock-shaft are such that the outer end of the guide-arm will 65 be at the farthest practicable distance from the rock-shaft at the beginning of the oscillation-that is, just in advance of the lower dead-center point of such arm or the line through its pivotal center and the center of 70 the rock-shaft.

The action of the parts is this: As the vibrating arm of the main shaft rises, the guidearm causes the joined ends of the links to move in a circle about its pivotal center, and 75 as that circle constantly approaches the circle through which the outer end of the driven rock-shaft crank moves, as seen in Fig. 1, it is obvious that the said crank must move constantly faster, and also through a greater angle 80 than the guide-arm; and hence that, when the guide-arm has moved one hundred and eighty degrees and reached the point where its outer end is nearest the center of the driven rockshaft, the crank-arm of such shaft will have 85 moved through a much greater angle than one hundred and eighty degrees. The extent of this angular motion of the driven rock-shaft is so much dependent upon the relative size and arrangement of the parts composing the 90 motion that its limits cannot be expressed for any but single constructions. However, with the mechanism from which the present drawings are made I have produced a very free and even oscillatory motion of the driven rock- 95 shaft through two hundred and seventy degrees and more.

I am aware that other means than the guidearms—such, for instance, as a fixed camgroove-may be used to control the motion of 100 the joint between the links; but I prefer the guide-arm shown, as with such construction all the parts move about centers, and hence I is a fixed block, set eccentrically to the with great freedom and evenness of motion,

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high rate of speed.

The application of this motion to drive a circularly oscillating shuttle in a sewing-ma-5 chine is shown and described in an application filed May 9, 1884, of which this is a division, and in which application due disclaimer of the herein-described matter is made by me.
What is claimed as new is—

1. As-a means for transmitting angular motion, the combination of a vibrating arm, a rock-shaft crank, two links jointed together, one of which is pivoted to said arm and the 15 other to said crank, and a guide arranged to cause the joint between the links to move

which permits of the parts being run at a very | along a curve other than that of the rock-shaft crank-pin, for the purpose set forth.

2. As a means for transmitting angular motion, the combination of a vibrating arm, a 20 rock-shaft crank, two links jointed together, one of which is pivoted to said arm and the other to said crank, and a guide-arm pivoted at one end to the joint between the links and at its other end to an axis arranged eccentri- 25 cally to the rock-shaft, substantially as and for the purpose set forth.

JAMES TRIPP.

Witnesses: JOHN BRICE, R. F. GAYLORD.